

$$E = \frac{\sigma}{\epsilon} \rightarrow \frac{f}{A} \cdot \frac{\Delta}{L} \Rightarrow \Delta = \frac{f \cdot L}{A \cdot E}$$

$$\begin{aligned} 1) \quad E &= 207 \text{ GPa} \\ d &= 10.2 \text{ mm} \\ F &= 8960 \text{ N} \\ l_0 &= ? \end{aligned}$$

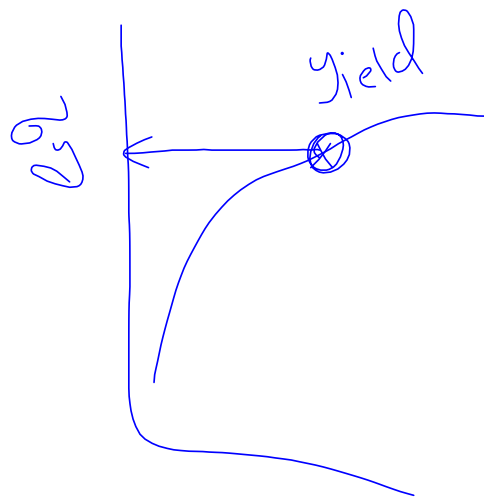
$$\begin{aligned} E &= \frac{\sigma}{\epsilon} \\ \Delta &= \frac{F \cdot L}{A \cdot E} \end{aligned}$$

$$0.25 = \frac{8900 \times L_0}{\frac{\pi}{4} (10.2)^2 \times 207 \times 10^3}$$
$$L_0 = 475 \text{ mm}$$

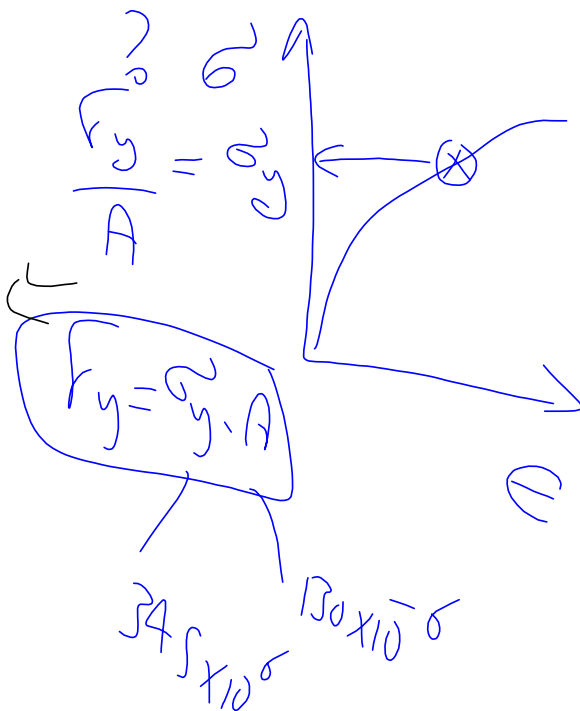
$$0.25 \times 10 = \frac{-3 \times 8900 \times L_0}{\frac{\pi}{4} (10.2 \times 10^{-3})^2 \times 207 \times 10^9}$$
$$\Rightarrow L_0 = 0.475 \text{ m}$$

2)

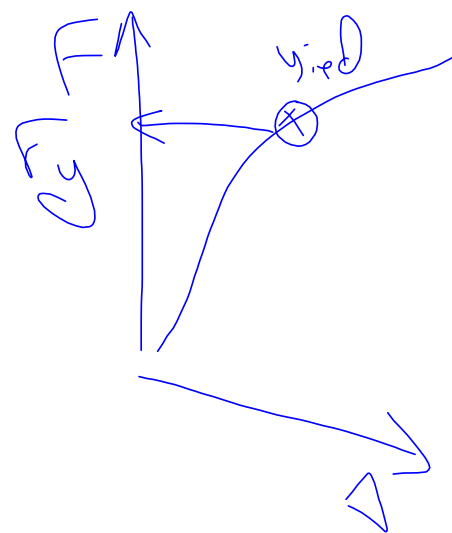
$$\sigma_y = 345 \text{ MPa}$$
$$E = 103 \text{ GPa}$$
$$A = 130 \text{ mm}^2$$
$$L_0 = 76 \text{ mm}$$



9)



find f_{max} at yield

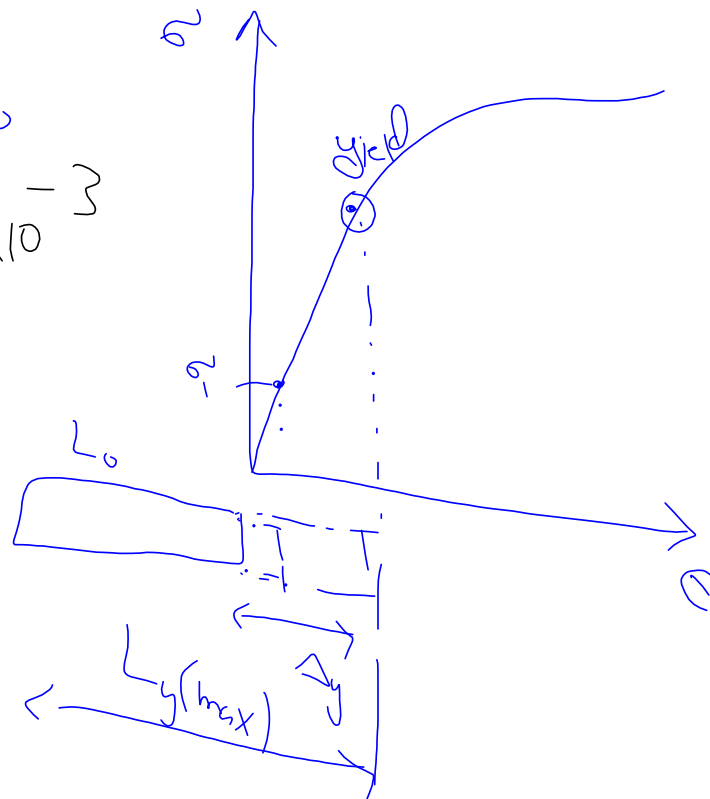


$$L_y = \Delta_y + L_0$$

$$\Delta_y = \frac{F_y L_0}{A \cdot E} \sim 76 \times 10^{-3}$$

$$130 \times 10^{-6} \quad 103 \times 10^9$$

$$L_y = L_0 + \Delta_y$$



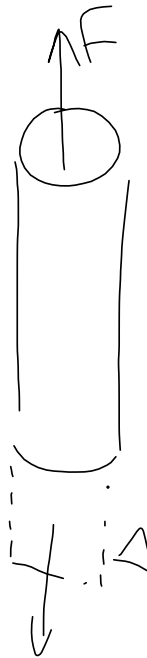
3)

$$d = 15.2 \text{ mm}$$

$$L_0 = 250 \text{ mm}$$

$$F = 48,900 \text{ N}$$

$$\Delta = ?$$



$$E = \frac{\sigma}{\epsilon}$$

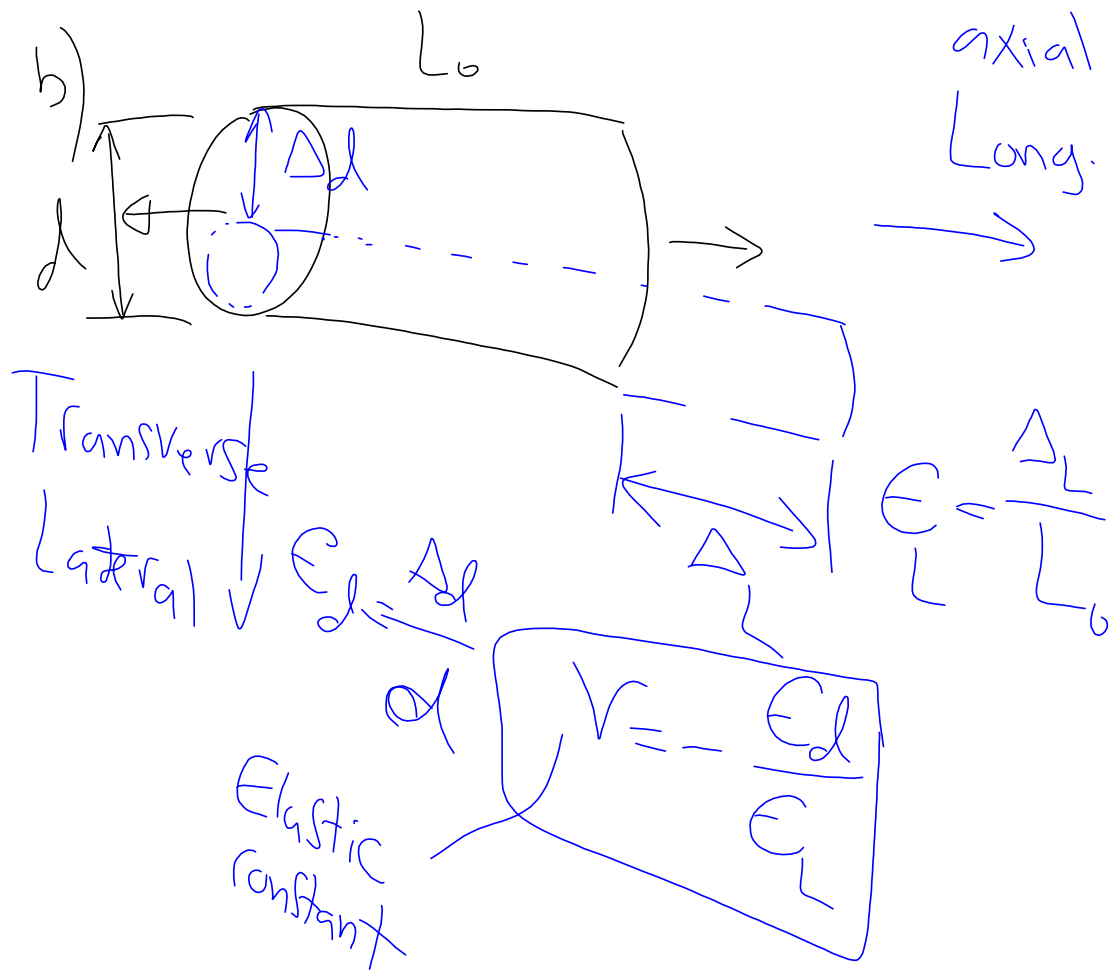
$$\Delta = \frac{F \cdot L_0}{A \cdot E}$$

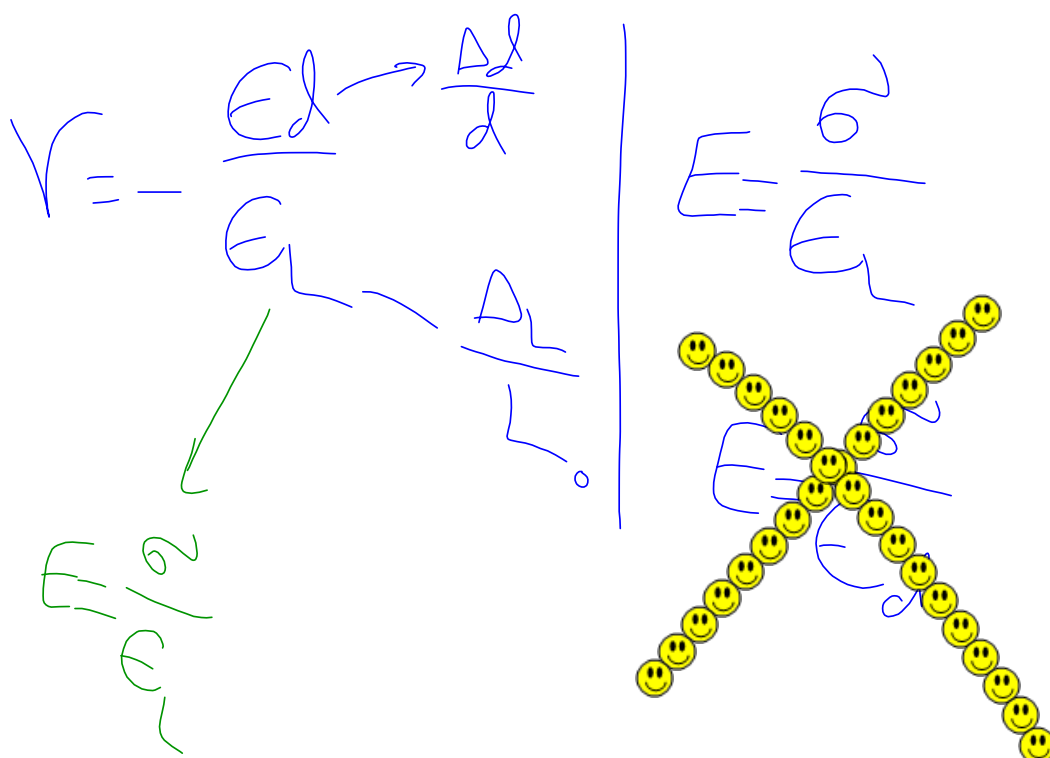
$$48,900 \text{ N}$$
$$\Delta = \frac{F \cdot L_0}{A \cdot E} = 0.25 \text{ m}$$

a)

$$\frac{\pi}{4} (15.2 \times 10^{-3})^2$$

Table 6.1





Elastic constants

E, ν, G

$$E = 2G(1 + \nu)$$

→ Shear Modulus or
Modulus of Rigidity

$$\begin{aligned}c) \quad E &= 2G(1+r) \\ 2076Pa &= 2G(1+0.3) \\ G &\approx 6P_s\end{aligned}$$

4)

$$\sigma_y = 249 \text{ MPa}$$

$$\text{T.S.} = 310 \text{ MPa}$$

$$E = 110 \text{ GPa}$$

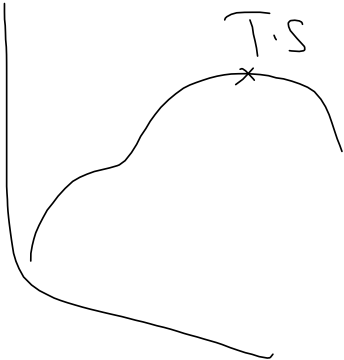
$$d = 15.2 \text{ mm}$$

$$L_0 = 300 \text{ mm}$$

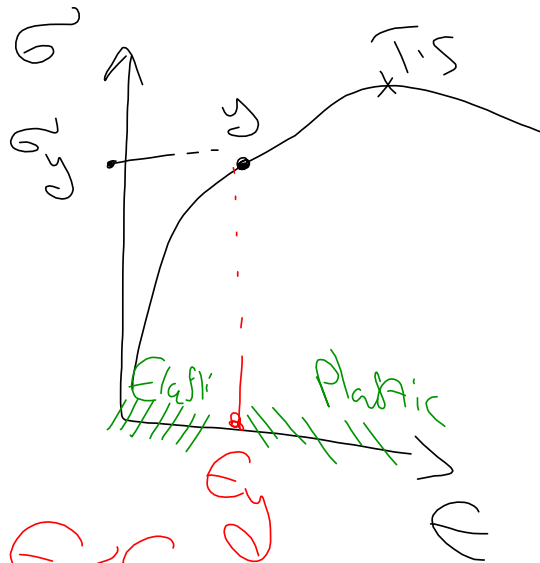
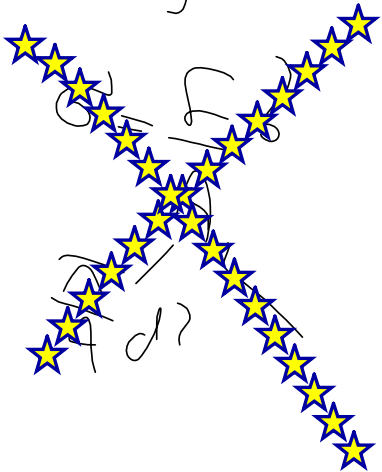
Tensile load

$$D = 1.9 \text{ mm}$$

load?



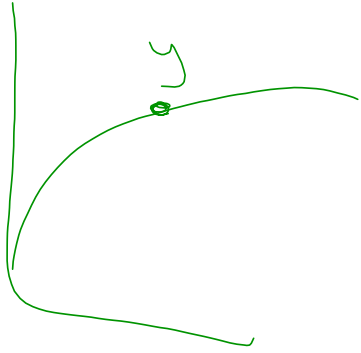
$\sigma < \sigma_y \Rightarrow \text{Elastic}$
 $\sigma > \sigma_y \Rightarrow \text{Plastic}$



$\sigma < \sigma_y \Rightarrow \text{Elastic}$
 $\sigma > \sigma_y \Rightarrow \text{Plastic}$

$E_y = ?$
 $E = \frac{\sigma}{\epsilon} \Rightarrow E_y = \frac{\sigma_y}{\epsilon_y}$
 $E = \frac{\Delta L}{L_0} = \frac{1.9}{300}$

249×10^6
 110×10^9



Since $E > E_y \rightarrow$ Plastic \rightarrow (ANNOI FIND F

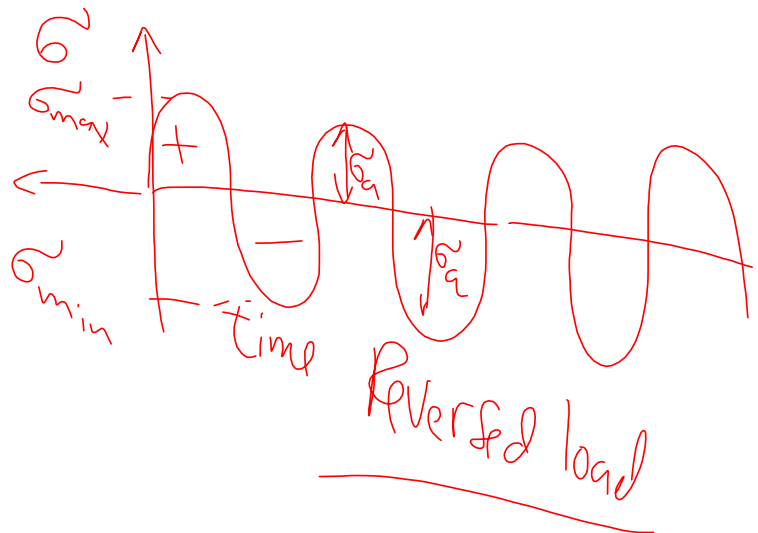
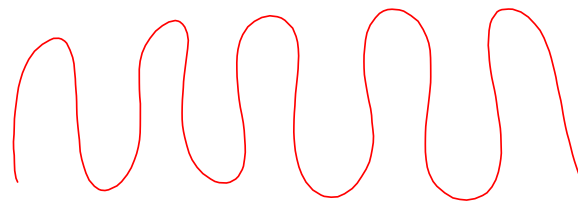
fatigue

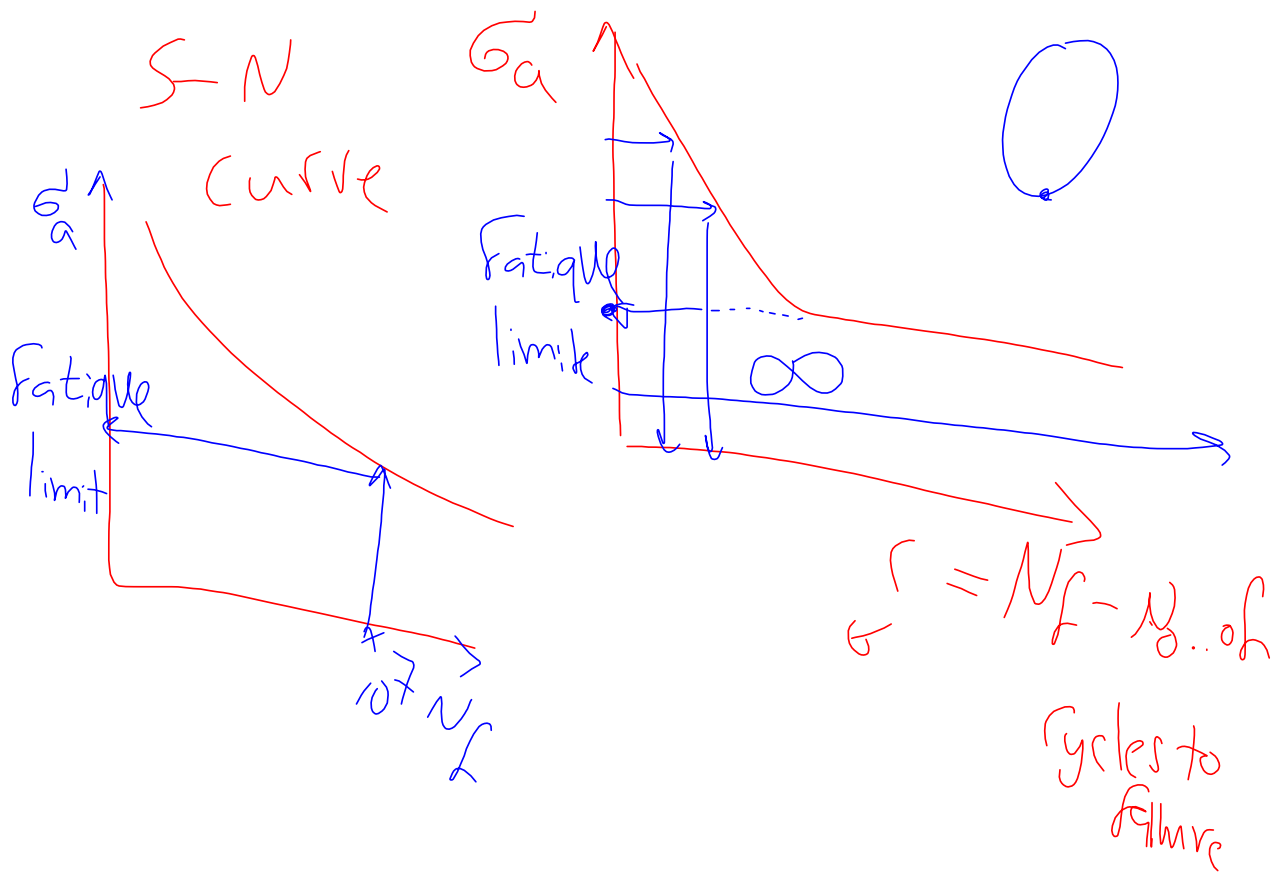
$$\sigma_m = \frac{\sigma_{max} + \sigma_{min}}{2}$$

mean

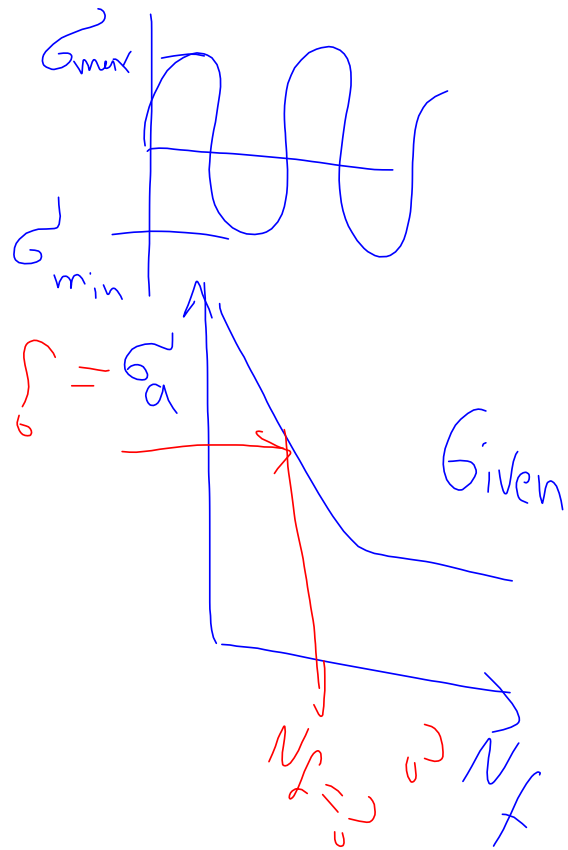
$$\sigma_a = \frac{\sigma_{max} - \sigma_{min}}{2}$$

amplitude





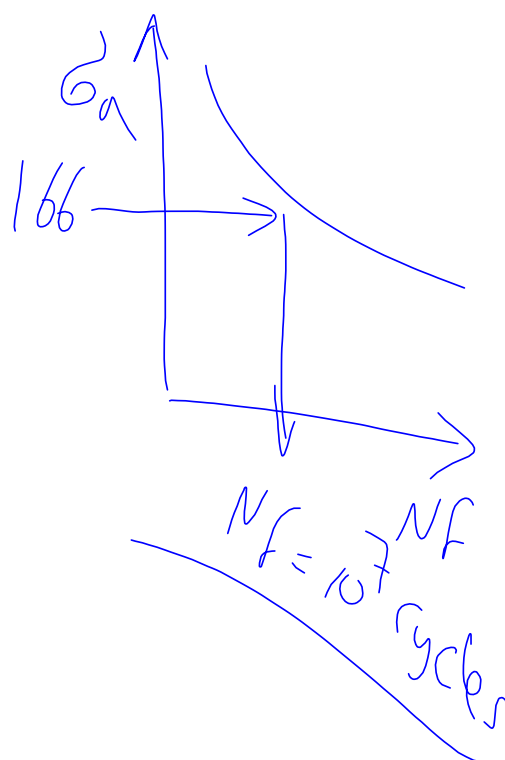
1) $d = 6.4 \text{ mm}$
Reverse
 $f_{max} = +5340 \text{ N}$
 $f_{min} = -5340 \text{ N}$
 $N_f = ?$
 0



$$\sigma_a = \frac{\sigma_{max} - \sigma_{min}}{2} = \underline{\underline{166 \text{ MPa}}}$$

$$\sigma_{max} = \frac{F_{max}}{A}, \quad \sigma_{min} = \frac{F_{min}}{A} = -5430$$

$$\sigma_a = \frac{F_a}{A} \quad \Rightarrow \quad F_a = \frac{F_{max} - F_{min}}{2} \quad \Rightarrow \quad \frac{\Delta}{4} d^2$$



b) $Sf = 1.5$

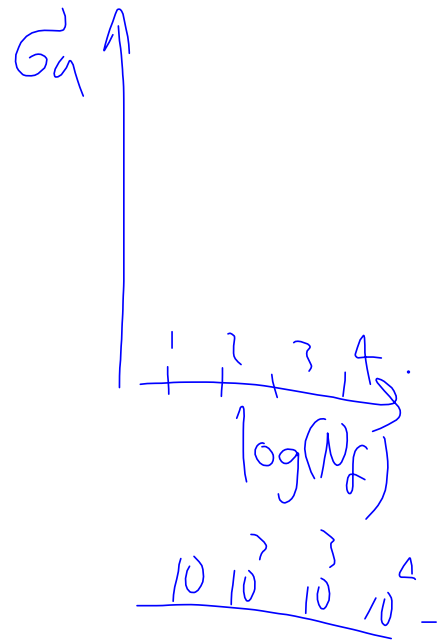
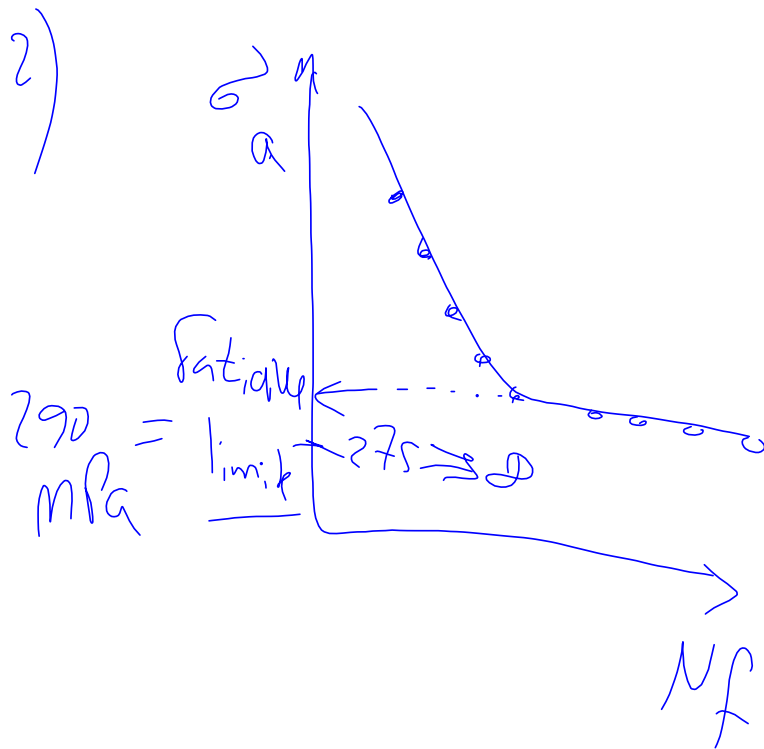
$$\sigma_a = \frac{\sigma_a}{Sf} = \frac{166}{1.5} = \underline{111 \text{ MPa}}$$

New

design

allowable

$$\rightarrow N_f = \underline{10^9 \text{ cycles}}$$

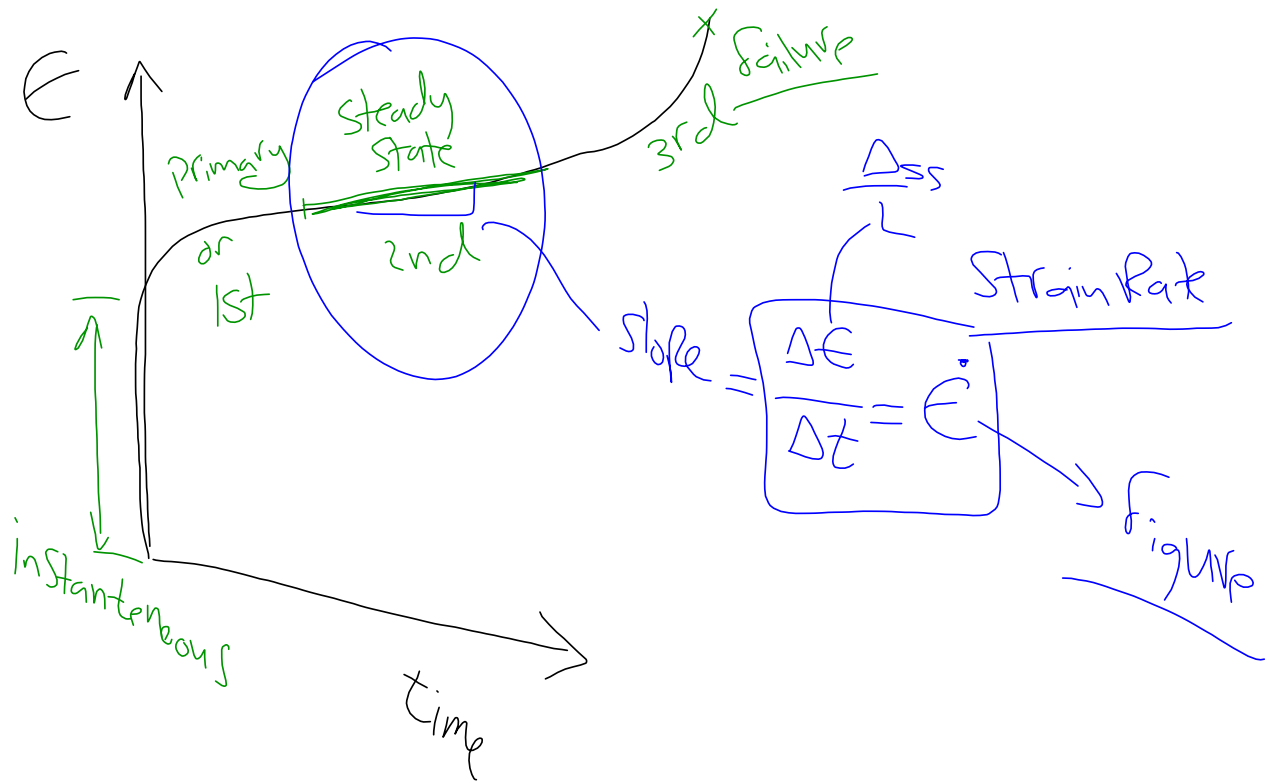


Creep ?

Time dependent
load is constant

Temp $>$ 40% T_m

↓
melting



1) $L = 1015 \text{ mm}$
 $\sigma = 70 \text{ MPa}$
 $T = 427^\circ \text{C}$

$\Delta = \Delta_{\text{total}}$ time = 10,000 hr

$\Delta_{\text{inst}} + \Delta_{\text{inst}} = 1,5$

Primary \rightarrow

$$D_{total} = D_{SS} + \underbrace{(D_{inst.} + D_{1st})}_{1.3}$$

$$\dot{E} = \frac{E}{time} = \frac{E}{10,000}$$

$$\dot{E} = \frac{\Delta_{SS}}{10/s}$$

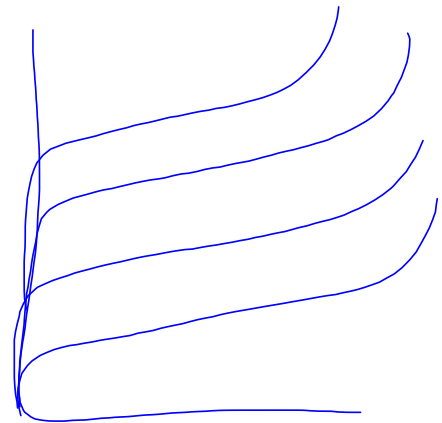
$$\dot{E} = 47 \times 10^{-7}$$

$$\sigma = 70 MPa \quad 427 C^{\circ}$$

$$\text{Strain } \epsilon = \frac{\Delta}{L_0}$$

2) $E_i = ?$

Temp = 1300K
 $\sigma = 83 \text{ Pa}$



$E_s = k_2 \sigma \exp\left(-\frac{P_c}{RT}\right)$
 8.5 = 5
 ?
 $P_c = 1500$

$$6.6 \times 10^{-4} = k_2(140) \exp\left(-\frac{Q_c}{8.51 \times 1090}\right) \quad \text{--- (1)}$$

$$8.8 \times 10^{-2} = k_2(140) \exp\left(-\frac{Q_c}{8.51 \times 1700}\right) \quad \text{--- (2)}$$

Solve to get Q_c & k_2 → Sub.in

Ans.